BUILDING A MORLD OF DIFFERENCE

Case Study for Combined Cycle CHP CIBO

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Agenda

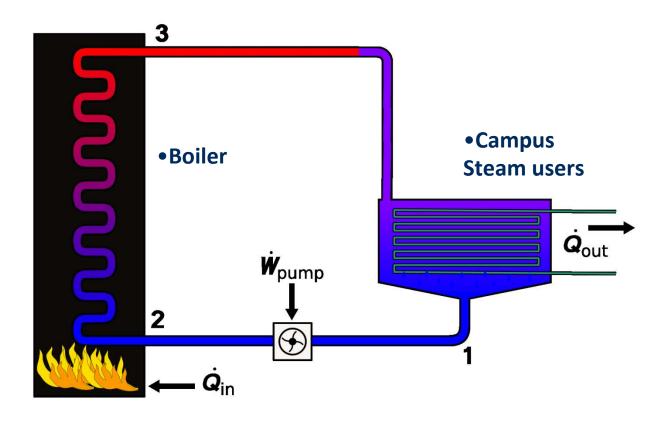
- 1. Overview of Central Heating Plants
- 2. Need for Power Generation
- 3. Case Study
- 4. Apply Combined Cycle
- 5. Questions

1855, when Michigan State University opened, all of the buildings were heated with wood fireplaces. A series of fires drove leaders to find a better solution to building heating.



- 1890, first central heating plant at MSU
- Steam district heating, coal fuel





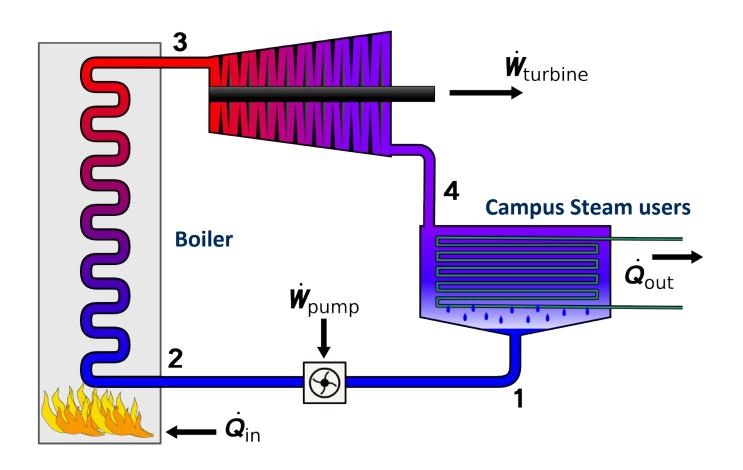
Basic Steam Heating Plant



- Rankine thermodynamic cycle converts heat to rotational energy by boosting steam pressure
- 1897, University of Michigan: 75 kilowatt,
 220 volt, backpressure electrical generator
- 1940, Cornell: 200 psi superheated steam,
 300 kW backpressure turbines
- Coal fuel

Cogeneration, or Combined Heat & Power (CHP)





Rankine Thermodynamic Cycle



Low pressure Rankine Cycle serves low electric load and high heating load

- First produce steam, the primary interest
- Second use steam to produce power on the way to serving <u>heating</u> loads

More than 20 lbs steam per kilowatt hour of electricity produced



Primary Purpose: Heat

- Electric loads: lights, radios, small fans
- Heat loads: poorly insulated, drafty buildings, domestic hot water

Low Electric Load, High Heating Load

Second half of 20th century: enter modern conveniences

• Television, clothes dryers, vacuum sweepers, air conditioners, toasters, electric stoves, microwave ovens, computers, and multiples of all these

Electric loads rising

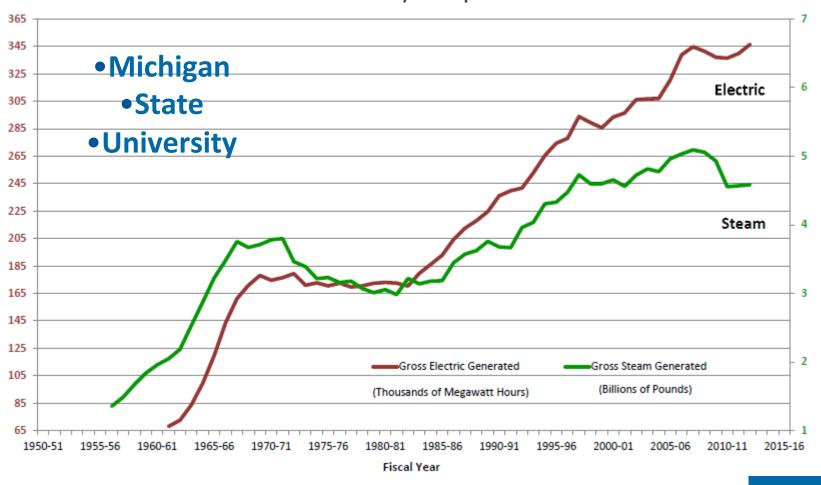
Second half of 20th century: Energy Crisis of 1974

- Building design: thermal insulation, double pane windows, weather stripping, ventilation heat recovery, energy efficiency standards
- Direct fired natural gas heat in industry
- Steam heating loads dropping

Need: More Power Generation, Less Steam

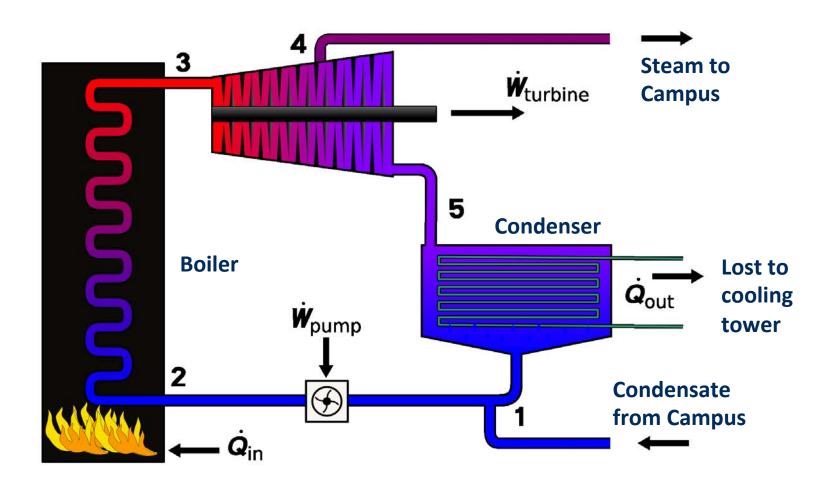


Infrastructure Planning and Facilities Power and Water Department Annual Utility Consumption



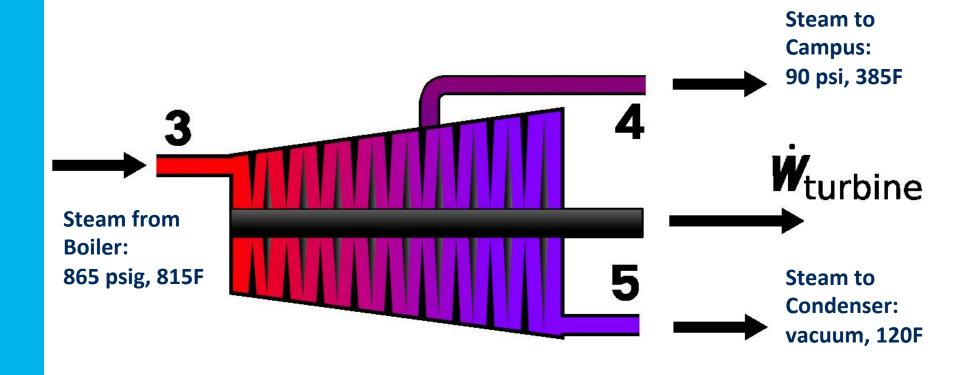
Michigan State University's T.B. Simon Power Plant

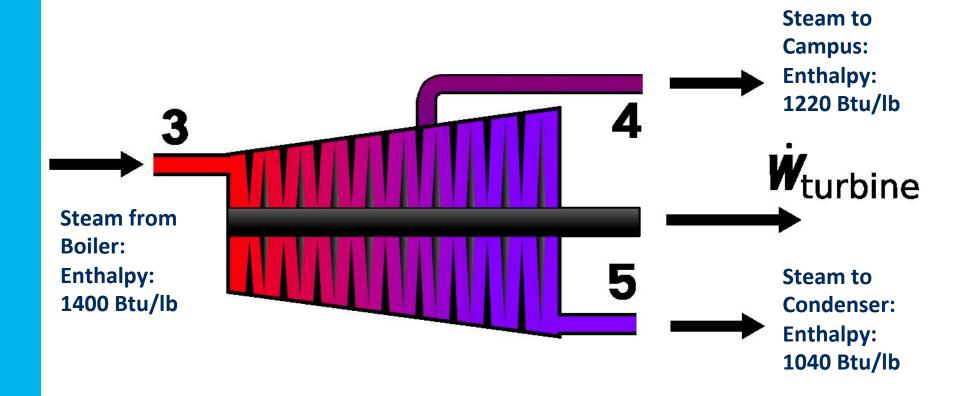
- Built in 1965
- Pulverized coal boilers: 865 psig, 815F superheated steam
- Condensing steam turbines with extraction
- Expanded with campus growth to serve all loads

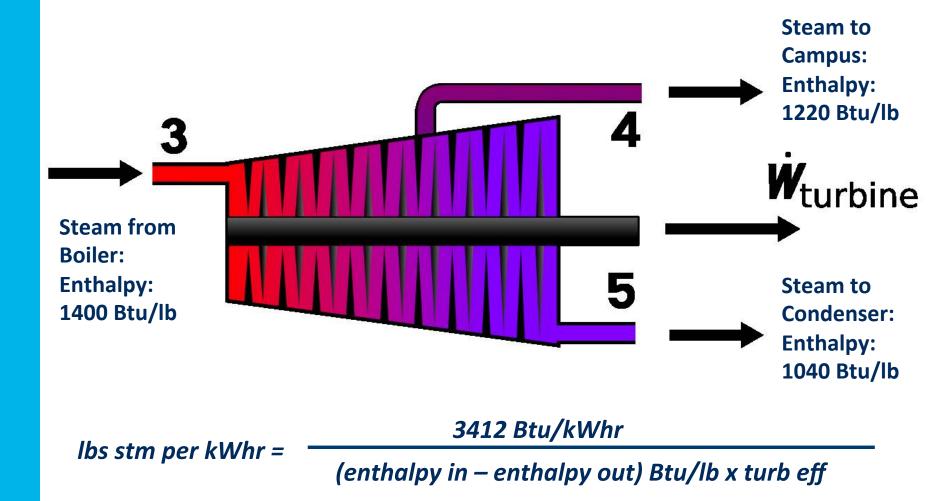


Condensing Steam Turbine with Extraction



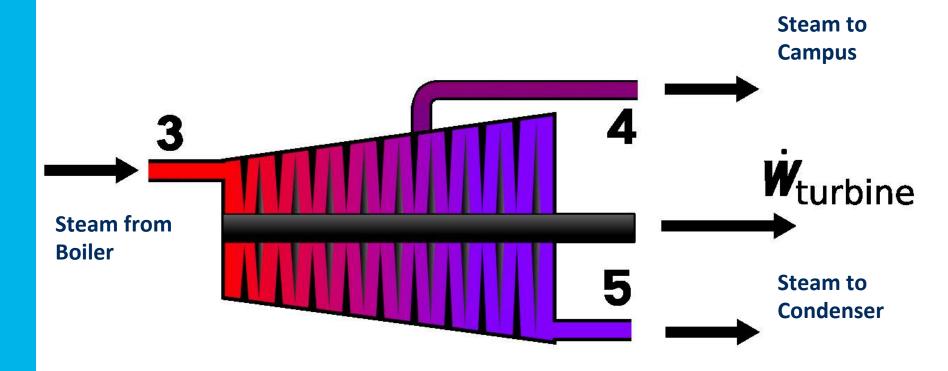






Condensing Steam Turbine Performance





Ibs steam per kWhr, point 3 to 4 = 20 lbs steam / kWhr

Ibs steam per kWhr, point 3 to 5 = 10 lbs steam / kWhr

Only 10 lbs steam per kilowatt hour of electricity produced through condenser, but

- The heat content of all of the 10 lbs of condenser steam per kWhr is delivered to the cooling tower
- Enthalpy diff of that steam minus condensate return, at 950 Btu/lb, is lost to atmosphere

Michigan State University's T.B. Simon Power Plant

Total plant net outputs in FY 2012:

Total steam produced: 4,582,000 klbs

Steam used on campus: 2,222,000 klbs

Generated power to campus: 235,000 MWhr

Clg tower condensed steam: 958,000 klbs

Michigan State University's T.B. Simon Power Plant

Steam Turbine Conditions in FY 2012:

- Steam turbine performance: 15 lbs steam/
 kWhr
- Actual Campus Energy Mix:

$$\frac{2,222,000 \text{ klbs}}{235,000 \text{ MWhr}} = 9.5 \text{ lbs / kWhr}$$

Michigan State University's T.B. Simon Power Plant

Total plant net thermal efficiency in FY 2012:

$$Eff = \frac{235,000 \text{ MWhrs } \times 3412 \text{ Btu/kWhr} + 2,434,000 \text{ MMBtu}}{5,794,000 \text{ MMBtu}}$$

= 56%

Energy Outputs Accounting FY 2012, in MMBtu:

Campus electricity: 802,000

= 16%

Plant electricity:

247,000 = 5%

Campus steam:

2,434,000 = 47%

• Plant steam:

650,000 = 13%

Clg tower steam:

<u>980,000</u> = <u>19%</u>

Total:

5,113,000 = 100%

In order to improve efficiency, consider decreasing:

M	M	Btu

• Plant electricity: 247,000 = 5%

• Plant steam: 650,000 = 13%

• Clg tower steam: 980,000 = 19%

Total: 1,877,000 = 37%

Consider a Combined Cycle CHP

Thermodynamic reasons:

- CTG makes power without any steam
- Less steam means less DA and feedwater heating steam, less water treatment
- Less steam power means less cooling tower
- Eliminates fans, pulverizers, coal handling, ash handling loads



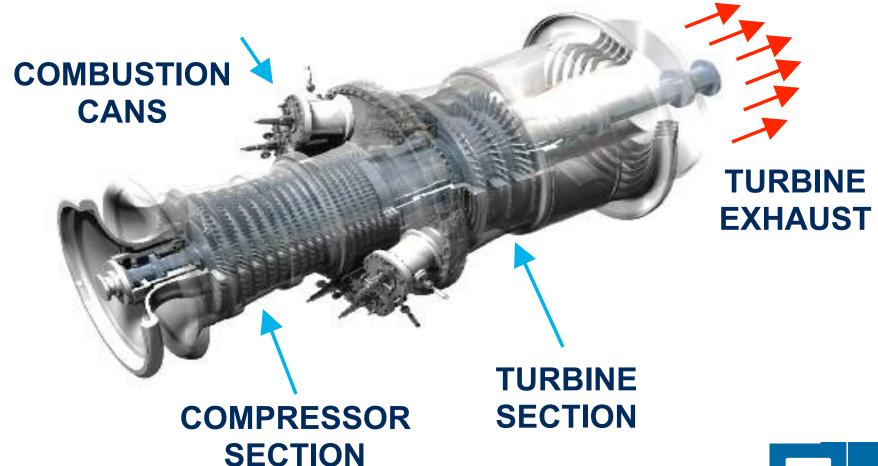
Major Equipment

- Combustion Turbine Generator
- Heat Recovery Steam Generator
- Steam Turbine Generator
- Fuel gas compressor, if needed

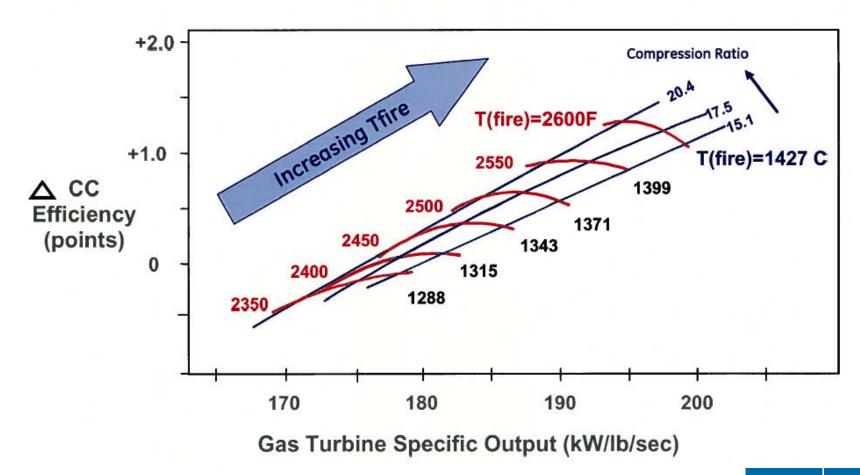
Cycle Sequence

- First, generate electricity in combustion turbine generator
- Second, generate steam in heat recovery steam generator
- Third, generate more electricity in steam turbine generator
- Fourth, send steam to campus

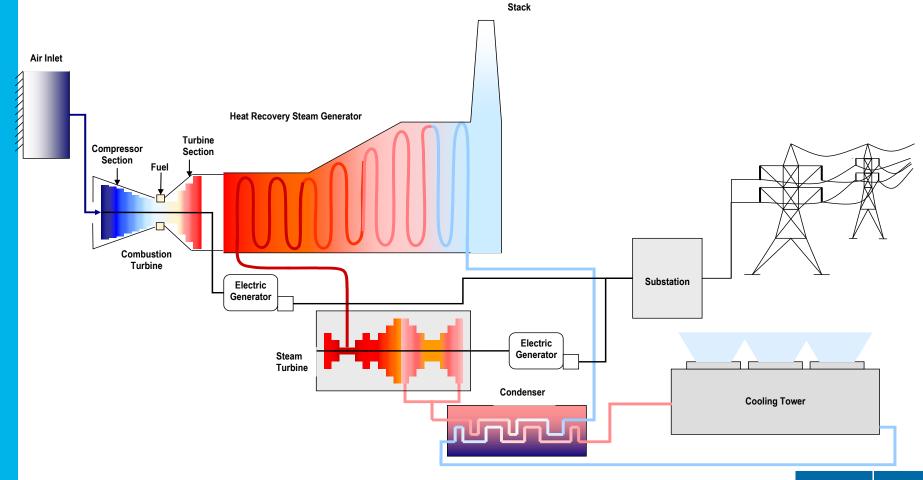
Combustion Turbine Generator



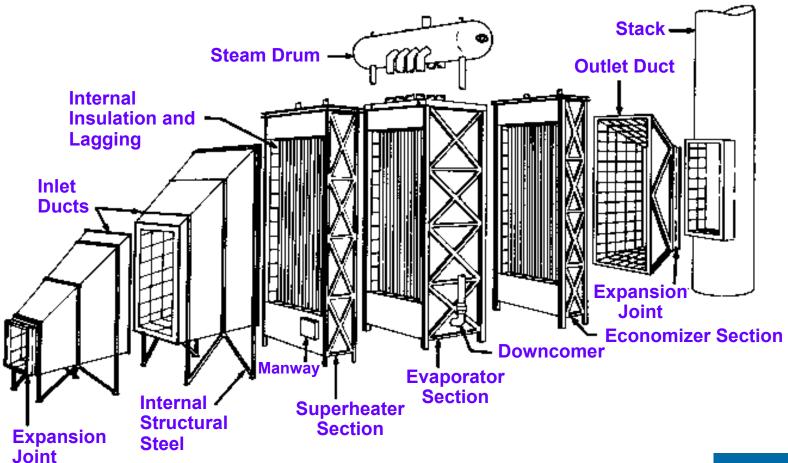
Combustion Turbine Inlet Temperature (Tfire)



Combined Cycle Process Flow



Heat Recovery Steam Generator (HRSG)



Deaerating Condenser or Pressurized Deaerator

- Combined cycle designs do not use regenerative feed water heaters
- Condensate supplied to the HRSG should be as cold as possible.
- Recirculation of hot condensate, already heated by the HRSG, will typically be used to raise the condensate temperature to 140 F before it enters the HRSG
- The HRSG can incorporate an integral pressurized deaerator onto the low pressure steam drum
- Alternatively, if there is minimal cycle makeup flow, a deaerating condenser is adequate.
- The boiler feed pumps take suction from the low pressure drum in both configurations

Michigan State University's T.B. Simon Power Plant

Apply a Solar Titan 250 to the FY 2012 data:

Steam used on campus: 2,222,000 klbs

Generated power to campus: 235,000 MWhr

Campus energy mix: 9.5 lbs steam/kWhr,

. . . and falling

Michigan State University's T.B. Simon Power Plant

Apply a Solar Titan 250 to the FY 2012 data:

Steam used on campus: 2,222,000 klbs

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. . . and falling

Solar Titan 250 Performance

Power gross output: 20,330 kW

Power net output: 19,010 kW

• Aux load: 1,320 kW

• Fuel input (NG HHV): 201,620 kBtu/hr

Net Power Efficiency: 32.2%

Net Power Heat Rate: 10,600 Btu/kWhr

Heat Recovery Steam Generator

Unfired steam output: 75,120 lb/hr

Unfired fuel input (NG HHV): 24,710 kBtu/hr

Unfired Steam turbine output: 3,760 kW

• Fired steam output: 157,200 lb/hr

• Fired fuel input (NG HHV): 125,650 kBtu/hr

Fired Steam turbine output: 7,860 kW

Combined Cycle Overall Performance

Unfired power net output: 22,770 kW

Unfired fuel input (NG HHV): 226,330 kBtu/hr

Unfired campus steam: 82,300 kBtu/hr

Fired power net output: 26,730 kW

• Fired fuel input (NG HHV): 327,260 kBtu/hr

• Fired campus steam: 172,220 kBtu/hr

Unfired Combined Cycle Overall Performance

$$Eff = \frac{22,770 \text{ kW } \times 3.412 \text{ kBtu/kWhr} + 82,300 \text{ kBtu/hr}}{226,330 \text{ MMBtu}} = 70\%$$

Steam Rate =
$$\frac{75,120 \text{ lb/hr}}{22,770 \text{ kW}} = 3.3 \text{ lb steam / kWhr}$$

Fired Combined Cycle Overall Performance

Eff =
$$\frac{•26,730 \text{ kW } \times 3.412 \text{ kBtu/kWhr} + 172,220 \text{ kBtu/hr}}{327,260 \text{ MMBtu}} = 80\%$$

Steam Rate =
$$\frac{157,200 \text{ lb/hr}}{26,730 \text{ kW}} = 5.9 \text{ lb steam / kWhr}$$

Conclusion

If you have CHP thermal-to-electric loads that are less than 15 lbs steam per kWhr of electricity, consider what a combustion turbine combined cycle CHP system could do to better balance your coincident energy outputs to your demands.

Questions?

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